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Novel heating/cooling stage designed for fluid inclusion 1 microthermometry of large stalagmite sections 2 Yves Krüger^{1,4,*}, Beat Hiltbrunner², Andres Luder¹, Dominik Fleitmann^{3,4,5}, Martin Frenz¹ 3 ¹Institute of Applied Physics, University of Bern, Switzerland 4 5 6 7 8 9 ²Astronomical Institute, University of Bern, Switzerland ³Institute of Geological Sciences, University of Bern, Switzerland ⁴Oeschger Centre for Climate Change Research, University of Bern, Switzerland ⁵present address: Department of Archaeology, School of Archaeology, Geography and Environmental Sciences, University of Reading, United Kingdom 10 11 12 13 Abstract 14 Liquid-vapour homogenisation temperatures of fluid inclusions in stalagmites are used for 15 quantitative temperature reconstructions in paleoclimate research. Specifically for this 16 application, we have developed a novel heating/cooling stage that can be operated with large 17 stalagmite sections of up to $17 \times 35 \text{ mm}^2$ to simplify and improve the chronological 18 reconstruction of paleotemperature time-series. The stage is designed for use of an oil 19 immersion objective and a high-NA condenser front lens to obtain high-resolution images for 20 bubble radius measurements. The temperature accuracy of the stage is better than ± 0.1 °C 21 with a precision (reproducibility) of ± 0.02 °C. 22 23 Keywords: microthermometry, heating stage, stalagmites, homogenisation temperature, 24 paleotemperature 25 26 **1. Introduction** 27 During past decades stalagmites have become an important archive providing 28 information on natural climate variations in the past (e.g. Fairchild et al., 2006; Lachniet, 29 2009; McDermott et al., 2004). Stalagmites can grow continuously over thousands of years 30 by precipitation of calcite from the cave drip water and their age can precisely be dated using 31 the uranium-series dating method. In weakly ventilated caves, the cave air temperature is 32 nearly constant throughout the year and is closely related to the mean annual surface 33 temperature outside the cave (Wigley and Brown, 1976). Thus, stalagmite formation 34 temperatures can be used to develop time series of paleotemperature variations. In recent

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35 years, different temperature proxies have been proposed to determine the formation temperatures of stalagmites: (i) the combination of δD and $\delta^{18}O$ of fluid inclusion water and 36 37 speleothem calcite, respectively (e.g. Schwarcz et al., 1976; McGarry et al., 2004; Vonhof et al., 2006; Zhang et al., 2008; Affolter et al., 2014), (ii) the excess of ¹³C-¹⁸O containing 38 39 molecules (so-called 'clumped isotopes') of calcite (Ghosh et al., 2006; Affek et al., 2008), 40 and (iii) the concentration of dissolved noble gases in the fluid inclusion water (Kluge et al., 41 2008; Scheidegger et al., 2010). 42 Recently, we have evaluated the applicability of fluid inclusion liquid-vapour 43 homogenisation temperatures as an alternative approach to determine stalagmite formation 44 temperatures (Krüger et al., 2011). The method relies on the measurement of the temperature 45 at which a two-phase liquid-vapour inclusion homogenises to a stable monophase liquid state 46 upon collapse of the vapour bubble. The temperature of the bubble collapse, i.e., the observed 47 homogenisation temperature $T_{h(obs)}$ depends on the density of the encapsulated drip water and 48 on the volume of the fluid inclusion. $T_{h(obs)}$ can be measured using a microscope 49 heating/cooling stage. Prior to measuring liquid-vapour homogenisation temperatures in 50 stalagmites, however, the initially monophase liquid inclusions have to be transferred to a 51 stable two-phase system by applying single ultra-short laser pulses to stimulate vapour bubble 52 nucleation in the metastable liquid (Krüger et al., 2007). Since stalagmites grow under 53 atmospheric pressure conditions, the density of the water preserved in the inclusions depends 54 only on the calcite formation temperature. By applying a thermodynamic model that accounts 55 for the effect of surface tension on liquid-vapour homogenisation (Marti et al., 2012) the 56 density of the encapsulated drip water and hence the formation temperature of the stalagmite can be calculated based on the measurement of $T_{h(obs)}$ and at least one additional measurement 57 58 of the vapour bubble radius at a known temperature. Previous studies on recent fluid 59 inclusions from the top part of actively growing stalagmites revealed a potential accuracy of 60 this new temperature proxy of approximately ±0.2 °C (Krüger et al., 2011). 61 62 2. Motivation to develop a novel heating/cooling stage 63 64 Fluid inclusions in stalagmites form during crystal growth (Kendall and Broughton, 65 1978) and are typically arranged along calcite growth bands. Thus, the inclusions and the 66 surrounding host calcite have the same age, which allows for a precise chronological 67 reconstruction of stalagmite formation temperatures determined from fluid inclusions. In

68 previous studies performed on a conventional Linkam THMSG 600 heating/freezing stage the

69 chronological reconstruction of the temperature data was complicated due to the fact that the 70 maximum sample size is limited to 7 mm in diameter. Hence, the initially ca. $20 \times 35 \text{ mm}^2$ 71 large and 300-400 µm thick stalagmite sections had to be broken into smaller pieces after 72 removing them from the carrier glass that is needed to stabilize the sample during cutting. 73 This means a considerable and time-consuming effort for sample documentation prior to 74 microthermometric measurements. Nevertheless a precise reconstruction of the sections often 75 failed when the sample fractured to tiny pieces upon dissolving the glue (cyanoacrylate) in an 76 acetone bath. Therefore, the main motivation to build a new heating/cooling stage was to be 77 able to examine entire stalagmite sections that no longer need to be removed from the carrier 78 glass but remain fixed on a 0.3 mm thick glass substrate with $28 \times 48 \text{ mm}^2$ standard 79 dimensions. 80 The second reason to design a new stage was the application of a microscope objective and a 81 condenser front lens with high numerical apertures (NA) to improve the optical resolution and 82 the quality of the vapour bubble images that are used for determining the bubble radius 83 (Spadin et al., in prep.). The bubble images are taken with a CCD camera at known sub-84 ambient temperatures: commonly at 5.1 °C, when the bubble radius reaches its maximum 85 (Marti et al., 2009). Since conventional heating/freezing stages like the Linkam THMSG 600 86 allow for microthermometric measurements from -180 up to 600 °C, their design requires the 87 application of long working distance (LWD) optics that, however, exhibit lower numerical 88 apertures than objectives and condenser front lenses with short working distances. For 89 example, the Olympus LMPlanFL 100x LWD objective we use for the Linkam stage has a 90 NA of 0.8, while the condenser front lens with a working distance of approximately 13 mm 91 has a NA of approximately 0.4. For the new heating/cooling stage that was designed for 92 applications in a narrow temperature range between -15 and 35° C, we wanted to use a 1.3 93 NA oil immersion objective (Olympus UPLFLN 100x) and a dry short working distance 94 condenser front lens with a NA of 0.9. The application of an oil immersion objective suggests 95 itself due to the fact that we do not polish the stalagmite sections but instead, we use 96 immersion oil to reduce light scattering at the rough surfaces and thus make the sections 97 transparent for microscopic observations in transmission. To illustrate the improvements of 98 the image quality and resolution, Fig. 1 shows a comparison of two microphotographs of the 99 same vapour bubble taken with the new (Fig. 1a) and the old (Fig. 1b) system. In Fig. 1a the 100 bubble image shows a dark ring with high contrast, whereas in Fig. 1b the dark ring appears 101 much broader and blurred due to lower contrast. Based on a simple ray-tracing model, Spadin 102 et al. (in prep.) found that reflection of light at the bubble meniscus results in a bright ring

surrounding the dark one. The intensity of this bright ring depends on the NA of the objective,

- 104 which is illustrated in Fig. 1c for the oil immersion objective (NA 1.3) and in Fig. 1d for the
- 105 LWD objective (NA 0.8). The intensity profile in Fig. 1c displays two distinct intensity
- 106 maxima outside the dark ring, whereas in Fig. 1d the two peaks can hardly be distinguished
- 107 from the background. In Fig. 1a and 1b the bright outer ring is only partially visible, which is
- 108 likely due shadow effects occurring in the stalagmite section and resulting in a non-uniform
- 109 illumination of the vapour bubble.
- 110 An additional advantage of the oil immersion objective compared to the LWD objective is the
- smaller focal spot size due to the higher numerical aperture and a significantly higher
- transmission of the 800 nm laser wavelength emitted from the Ti:sapphire femtosecond (fs)
- 113 laser that is used to induce bubble nucleation. This means that for a given energy and duration
- 114 of the fs-laser pulse, the oil immersion objective provides a much higher pulse intensity in the
- 115 focus than the LWD objective, and in consequence, allows us to induce vapour bubble
- nucleation in inclusions that are located up to 250 µm below the sample surface.



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118 Fig. 1: Comparison of bubble images: a) microphotograph using a 100x oil immersion

119 objective (NA 1.3) and a condenser NA of 0.9. b) microphotograph using a 100x LWD

- 120 objective (NA 0.8) and a condenser NA of 0.4. c) and d) intensity profiles along the dashed
- 121 lines indicated in a) and b). Arrows indicate the intensity maximum of the outer bright ring.
- 122

123 **3. Design of the heating/cooling stage**

- 124
- 125 In the following we describe the design of the novel heating/cooling stage that is

126 illustrated in Fig. 2a-e. All components described in the text are labelled in the figures 127 accordingly. The heating/cooling stage is mounted on a Olympus BX51 microscope (Fig. 2c) 128 and its dimensioning complies with the requirements set by the large sample size and the 129 short working distances of the microscope optics. To meet these demands we constructed a 130 large-sized (Ø 116 mm) heating/cooling block (A) made from copper, a material with 131 excellent thermal conductivity. The rear side of this copper block features a central recess to 132 bring the condenser front lens (C) close to the sample (Fig. 2b). The sample (III) is placed on 133 the top face of the block (Fig. 2a). The 0.9 NA condenser front lens used has a working 134 distance of ~3.5 mm and illuminates the sample through a tapered aperture with a final 135 diameter of 1.5 mm (Fig. 2b). The heating/cooling block (A) comprises (i) a sheathed twin-136 core heating conductor (1; Thermocoax) of 1.7 m length and a diameter of 1.5 mm, which is 137 soldered in a spirally routed groove closely below to the surface (Fig. 2b); (ii) a cooling unit 138 (II) consisting of a cooling circuit (2) with inlet and outlet connections for the cooling liquid, 139 a water-ethylene glycol mixture. The cooling unit is placed below the heating conductor; (iii) 140 a Pt-100 thin-film temperature sensor (3; Heraeus) embedded in a slot on the top face of the 141 copper block (Fig. 2a). 142 The sample (III), i.e., the 0.3 mm carrier glass with the stalagmite section, is fixed in a 2 mm 143 thick fibreglass frame that prevents leakage of the immersion oil on the copper block (Fig. 144 2a). For the preparation of the sample we use UV curable epoxy resins that are inert against 145 the immersion oil and can be applied at sub-ambient temperatures. Prior to the cutting of the 146 sections, the stalagmite sample is fixed on the carrier glass using a low-viscous resin (Epo-147 Tek OG603) that closely matches the refractive index of calcite ($n_{\epsilon} = 1.486$) in the cured state. 148 The carrier glass is then fixed to the bottom side of the fibreglass frame using a different, 149 higher viscous resin (Epo-Tek OG142). The fibreglass frame can be magnetically attached to 150 the sample holder (V) that is mounted on an x-y translation stage (IV). The translation stage is 151 moved by means of two micrometre gauges (4) with travel lengths of 50 mm and 25 mm, 152 respectively. The position of the sample can be read off from the scales on the micrometre 153 gauges, which allows for a precise localisation of the analysed fluid inclusions mandatory for 154 the chronological reconstruction of the temperature data. The maximum size of the stalagmite 155 sections that can be scanned with the oil immersion objective is $17 \times 35 \text{ mm}^2$. The application of an immersion objective (D) that is in direct thermal contact with the 156 157 oil film covering the stalagmite sample requires an active control of the objective temperature 158 to minimise vertical temperature gradients and to achieve faster equilibration of the sample

to minimise vertical temperature gradients and to achieve faster equilibration of the sample

 $159 \qquad \text{temperature. Therefore, we have developed a copper heating/cooling jacket (B) that allows for} \\$

160 a precise regulation of the objective temperature. To accomplish an efficient heat transfer, the 161 heating/cooling jacket is attached directly to the inner brass cylinder that houses the objective 162 lenses (D). To this end, we removed the outer protective sleeve of the objective and modified 163 the spring system that protects the front lens from mechanical damage. The cylindrical 164 heating/cooling jacket consists of a heater element (I) that is made up of a coiled twin-core 165 heating conductor (1) embedded between two copper sleeves and an outer cooling unit (II) 166 with inlet and outlet connections (2) for the cooling liquid. The heater element (I) is screwed 167 on the objective (D) and features a 2 mm spring deflection relative to the cylindrical cooling 168 unit (II) that serves as guidance for the spring suspension of the objective. An insulating 169 sheath (9) made from Polyoxymethylene (POM), finally, encases the heating/cooling jacket. 170 The back aperture of the POM sheath is sealed with a 1 mm glass window (10) to prevent 171 condensation of moisture on the objective rear lens and within the heating/cooling jacket. The 172 whole assembly is fixed to the microscope frame by means of a detachable single-position 173 nosepiece (VII) with centring screws to align the objective to the optical axis. A lateral 174 through-hole (11, Fig. 2d) in the nosepiece can be used to flush the glass window with 175 nitrogen gas to remove the condensed air moisture from the glass window. The objective 176 temperature, finally, is measured by a Pt-100 thin-film sensor (3) that is placed close to the 177 objective front lens (Fig. 2d). 178 The sample chamber is placed inside a sealed off aluminium housing (VIII) to 179 minimise condensation of air moisture and thermal effects induced by air circulation. An 180 opening in the base plate (Ø 34 mm) provides access to enter the condenser front lens (C) and 181 is sealed off by a V-seal (7) mounted on the outer face of the adapter tube (8) that tightly 182 connects the aperture diaphragm and the front lens of the condenser (Fig. 2e). From above, 183 the objective with the heating/cooling jacket can be entered through an opening (\emptyset 60 mm) in 184 the centre of the detachable cover plate (5). A V-seal (7) mounted on the POM insulation 185 sheath seals off the port when the objective comes close to the sample. The cover plate is 186 fixed to the housing by means of two clamping levers (6) and thus can be easily and quickly 187 removed to load the sample. Finally, a plug board (VI) in the side wall of the aluminium 188 housing is used as lead-through to connect the lower heating/cooling block (A) to the electric 189 and the hydraulic circuits (Fig. 2a,c).

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190 191

192Fig. 2: a) View of the heating/cooling stage with removed cover b) Cross-section through the stage and the193objective heating/cooling jacket c) stage mounted on the microscope d) objective with heating/cooling jacket e)

- condenser assembly.
- 195 Legend:
- A: heating/cooling block B: objective heating/cooling jacket C: condenser lens D: oil immersion objective.
- 197 I: heater element II: cooling element III: sample IV: x-y translation stage V: sample holder VI: plug board VII:
 198 objective nose piece VIII: stage housing IX remote control.
- 199 1: heating conductors 2: cooling circuits 3: Pt-100 sensors 4: micrometer gauge 5: detachable cover plate 6:
- 200 clamping lever 7: V-seal 8: adapter tube 9: insulating sheath 10: window 11:gas inlet
- 201 Connections are indicated by arrows: red for heating conductor, blue for cooling circuit, green for Pt-100 sensor202 and grey for nitrogen gas.
- 203 204

205 **4. Mode of operation of the stage**

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207 The twin-core heating conductors (1) in the lower copper block (A) and in the

- 208 objective jacket (B) are connected to separate power supply units (Elektro Automatik EA-PS
- 209 3150-04 B) to enable independent temperature control of the two heating/cooling elements. A
- 210 cryo-thermostat (Lauda ProLine RP 845) is used to feed the two cooling circuits and provides
- 211 for a constant temperature and flow rate of the cooling liquid. The stage is operated under
- 212 permanent circulation of the coolant, while the temperature of the two heating/cooling

213 elements is regulated individually by varying the voltage impressed on the heating

- 214 conductors. The stage reaches a maximum sample temperature that is 50°C in excess of the
- 215 coolant temperature set at the thermostat. Considering the specific application to fluid
- 216 inclusions in stalagmites and the use of immersion oil, a 50 °C temperature range of the stage
- 217 is more than sufficient. The two Pt-100 sensors (3) attached to the lower heating/cooling
- 218 block and to the microscope objective, respectively, are connected to a four-channel
- 219 temperature converter (National Instruments). The temperature converter and the two power
- 220 supply units are connected to a computer via USB interfaces. A LabView[®] based PID
- 221 (Propotional-Integral-Derivative) controller is used to regulate the temperature of the two
- 222 heating/cooling elements. For convenience, the stage can be operated via the function keys of
- 223 a remote control (IX, Fig. 2c). A flow chart of the temperature control system is shown in Fig.
- 224 225

3.

- Objective jacket

 Constant temperature cooling circuit

 Heating conductor

 Pi-100 sensor T₂ calibration

 Power supply

 Heating Conductor

 Pi-100 sensor T₂ calibration

 Power supply

 Heating Conductor

 PiD controller (LabView)

 Remote control

 Remote control
- Fig. 3: Scheme of the temperature control of the heating/cooling stage. For the calibration of the stage additional
 Pt-100 sensors can be connected to measure the sample temperature and to monitor room temperature variations
 (see section below)

233 5. Thermal characterisation and temperature calibration of the stage

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To characterise the thermal properties of the heating cooling stage such as precision,

- accuracy and equilibration times, we used an additional Pt-100 thin-film sensor fixed on a
- 237 0.3 mm glass substrate and embedded in epoxy resin (Fig. 4a). This sensor allows us to
- 238 measure the sample temperature (T_3) in the centre of the lower copper block, i.e., in the
- 239 optical path of the microscope. The regular Pt-100 sensor (T_l) embedded in the lower copper
- block is about 40 mm off the centre, which means that T_3 is not equal T_1 due to lateral thermal
- 241 gradients. The same applies for the objective temperature T_2 that is not equal to the

- temperature of the front lens T_4 (Fig 4b). Since we cannot directly measure the sample and front lens temperatures during fluid inclusion microthermometry, we need a calibration to determine T_3 and T_4 based on the two set temperatures T_1 and T_2 . Furthermore, T_4 should equal T_3 to avoid vertical temperature gradients within the sample. We recall that the objective front lens is in contact with sample via the immersion oil. The PID controller is capable of holding T_1 and T_2 constant within ±0.01 °C, which determines the fluctuation of T_3 and T_4 .
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Fig. 4: Positions of temperature measurements for the stage calibration. a) T_1 : surface temperature of the copper block 40 mm apart from the sample position, measured with a permanent Pt-100 sensor. T_3 : sample temperature measured with an additional Pt-100 sensor used only for calibration. b) T_2 : objective temperature close to the front lens measured with a permanent Pt-100 sensor. T_4 : temperature of the objective front lens, not measured directly (see text for details).



Fig. 5: Schematic representation of the measurement procedure used to calibrate of the objective front lens temperature T_4 by minimising the temperature gradient between T_3 and T_4 (see text for details)

263 For the calibration of the stage it was not practicable to measure both the front lens 264 temperature T_4 and the sample temperature T_3 independently. Instead of measuring T_4 265 directly, we only measured T_3 as a function of T_1 and minimised the vertical temperature 266 gradient between T_3 and T_4 by adjusting the objective set temperature T_2 . In practice, we 267 measured T_3 , first, in a non-contact configuration with the objective front lens about 0.3 mm 268 above the immersion oil covering the embedded Pt-100 sensor and, subsequently, in a contact 269 configuration with the front lens immersed in the oil and thus in thermal contact with the Pt-270 100 sensor. In this way, we were able to detect thermal gradients between the objective front 271 lens and the sample (Pt-100 sensor) on the basis of the temperature change of T_3 . By adjusting 272 the objective set temperature T_2 in repeated measurements, we finally minimised the thermal 273 gradients and thus the change of T_3 when switching over to the contact configuration. This 274 calibration procedure is illustrated schematically in Fig. 5. Calibration measurements were 275 performed at different sample temperatures T_3 and for different coolant temperatures $T_{coolant}$. 276 The results are shown in Fig. 6. The diagram displays the deviation of the two set 277 temperatures, T_1 and T_2 , from the calibrated sample temperature $T_{3(cal)}$ ($\Delta T_{set} = T_{set} - T_{3(cal)}$) as 278 function of $T_{3(cal)}$ for coolant temperatures of 0, -5, -10 and -15 °C. The relation between 279 ΔT_{set} and $T_{3(cal)}$ is nearly linear for both, the lower copper block (T_1) and the objective (T_2) . To 280 test our calibration, we repeated the measurements of the sample temperature in the contact 281 configuration. Measurements were taken in 5 °C increments using a heating/cooling rate of 5 282 °C/min and an equilibration time of 15 minutes after each temperature step. Fig. 7 illustrates 283 that the deviation of the measured sample temperatures $T_{3(meas)}$ from the calibration 284 temperature $T_{3(cal)}$ is less than ± 0.02 °C, except for some measurements performed with a 285 coolant temperature of -15 °C, which show a slightly larger deviations above 15 °C. This is 286 due to the limited cooling capacity of the thermostat, which results in a slight increase of 287 $T_{coolant}$ with increasing stage temperature and larger temperature fluctuations. In addition, Fig. 288 7 indicates a small hysteresis between heating and cooling runs. Repeated measurements with the same coolant temperature yield a precision (reproducibility) of T_3 of ± 0.01 °C. 289 290 The temperature tolerance of the Pt-100 sensors ($^{1}/_{3}$ Class B) used for the calibration is 291 ±0.1 °C at 0 °C and ±0.15 °C at 30 °C. To further improve the accuracy of the sample 292 temperature T_3 below $\pm 0.1^{\circ}$ C, we used synthetic H₂O and H₂O-CO₂ fluid inclusions for 293 absolute temperature calibrations. The use of synthetic fluid inclusions allows us to compare 294 our measurements with well-known reference temperatures, namely the melting temperature 295 of ice at 0.0 °C and the critical homogenisation of CO₂ in the H₂O- CO₂ system at 31.42 °C 296 (Morrison 1981). Measurements were preformed with different coolant temperatures yielding

297 a precision of ± 0.02 °C and a slight offset of T_3 to lower temperatures: -0.1 ° at 0.0 °C and -

298 0.15 ° at 31.42 °C. Using these offset values for a correction of the calibration, we can

achieve a temperature accuracy of the heating/cooling stage of approximately ± 0.05 °C.

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Fig 6: Deviation of the set temperatures T_1 and T_2 from the calibrated sample temperature $T_{3(cal)}$ for different coolant temperatures indicated by colours (see text for details).



Fig. 7: Calibration check illustrating the deviation of the measured sample temperature value $T_{3(meas)}$ from the calibration $T_{3(cal)}$. Filled symbols denote measurements upon heating, open symbols upon cooling. Colours indicate different coolant temperatures.

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302 The measurements of the ice melting temperature have also shown that absorption of 303 light in the sample can increase T_3 by up to 0.05 °C, in spite of using a heat-absorbing IR-304 filter (Schott KG5 IR) in the illumination light path. In consequence of the absorption the 305 effective sample temperature is higher than predicted by the calibration, which results in an 306 underestimation of the ice melting temperature. To deal with this potential uncertainty we are 307 working with a pre-defined setting of the light intensity and of the field diaphragm for sample 308 illumination. Finally, we investigated the effect of room temperature fluctuations on the 309 sample temperature. Both T_3 and the room temperature were measured with Pt-100 sensors. 310 The results of these measurements indicate that a change in room temperature of 1 °C affects 311 the sample temperature by less than 0.01 °C, which is a negligible error in an air-conditioned 312 lab with room temperature fluctuations of less than ± 2 °C. 313 Besides precision and accuracy we also analysed the dynamic properties of the stage, 314 namely the temperature equilibration times. The equilibration time t_{eq} denotes the time lag of

- 315 the system to reach a stable temperature state after a temperature change ΔT . The time at
- 316 which the set temperature reaches its new value serves as reference time t_0 . Equilibration
- 317 times were measured for T_1 , the temperature of the lower heating/cooling block, for T_2 , the

318	objective temperature and for the sample temperature T_3 . Figure 8 illustrates the nominal
319	(dashed lines) and the measured temperature trends (solid lines) of T_1 , T_2 and T_3 as a function
320	of time for different temperature changes ΔT and heating rates. The diagrams show that T_I
321	and T_2 slightly over-shoot the set temperatures after heating is stopped and then return to their
322	set values. The equilibration times t_{eq} of T_1 and T_2 are very similar and depend only on the
323	heating rate (~10 s at 1 °C/min, ~20 s at 10 °C/min). The sample temperature T_3 , in contrast,
324	approaches its set temperature asymptotically from below with a considerable time lag. The
325	time the system needs to reach a stable sample temperature is too long for practical use (up to
326	360 seconds) and therefore we defined three time markers t_1 , t_2 and t_3 at which the sample
327	temperature is 0.1, 0.05 and 0.02 °C, respectively, below the final sample temperature $T_3(eq)$.
328	Fig. 8 shows that t_1 , t_2 and t_3 depend on the heating rate as well as on the temperature change
329	ΔT . With respect to fluid inclusion measurements t ₁ implies a temperature overestimation of
330	0.1 °C, t ₂ of 0.05 °C and t ₃ of 0.02 °C. This means that the accuracy of the T_h measurements
331	also depends on the measuring routine that needs to be adjusted accordingly.
332	In practise, we use a rate of 10 $^{\circ}$ C/min to heat the stage to a temperature that is
333	approximately 2–3 °C below the expected homogenisation temperature. After an equilibration
334	time of 60-90 seconds, further heating is accomplished with a rate of 1 $^{\circ}C/min$. Close to the
335	homogenisation, finally, when the vapour bubble becomes very small, the temperature is
336	increased stepwise in 0.05 °C increments until T_h is reached. During this last phase of the
337	measurement the sample temperature closely follows the set temperature.
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220	



342 Fig. 8: Temperature-time plots illustrating the equilibration times of T_1 , T_2 and T_3 after temperature increases: a) 343 5 °C at a heating rate of 1 °C/min b) 5 °C at a heating rate of 10° C/min and c) 10 °C at a heating rate of 344 10 °C/min. Dashed lines indicate the set temperatures and solid lines represent the measured temperature trends.

- 345 The reference time t_0 denotes the end of the heat-up phase (dashed vertical line).
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- 347

6. Conclusions

351		
352		Liquid-vapour homogenisation of fluid inclusions in stalagmites is a promising new
353		approach to accurately reconstruct paleotemperature variations with high temporal resolution.
354		For this specific application, we have developed a novel heating/cooling stage for operation
355		with large stalagmite sections and high-NA optics. With a temperature accuracy well below
356		± 0.1 °C and a precision of ± 0.02 °C the stage meets the requirements for fluid inclusion
357		measurements in this temperature range. In a next step of development we will factor the
358		dynamic properties of the stage into the PID temperature control to further reduce the
359		equilibration times after large temperature changes and thus, to increase the efficiency of the
360		T_h measurements.
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363		
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368		environmental reconstructions from stalagmites from Switzerland, Turkey, Arabia and India"
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